

Code No: R05412101

R05**Set No. 2**

IV B.Tech I Semester Examinations, November 2010
THEORY OF VIBRATIONS AND AEROELASTICITY
Aeronautical Engineering

Time: 3 hours

Max Marks: 80

Answer any FIVE Questions
 All Questions carry equal marks

1. Figure below shows an equivalent model for a two storey building. The system consists of two rigid masses m_1 and m_2 supported on elastic columns. Columns as massless beams supported at both the ends. Elastic properties of the columns are as shown in the figure 6, Assume that the building only goes under horizontal motion. Let $m_1 = m_2 = m$, $H_1 = H_2 = H$, $I_1 = I_2 = I$.

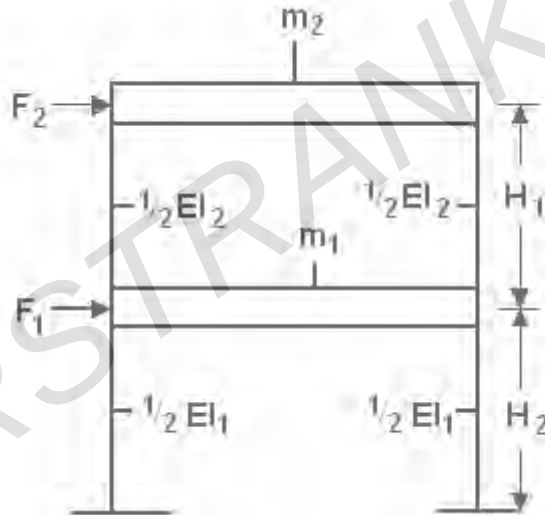


Figure 6

- (a) Derive the differential equations for the horizontal motion of the masses
 - (b) Obtain the natural frequencies of the system
 - (c) Obtain the natural modes of vibration
 - (d) Plot the modes. [16]
2. Describe the following:
- (a) D'Alembert's principle
 - (b) Various forms of damping
 - (c) Principle of virtual work
 - (d) Amplitude vs Frequency response of a spring-mass-damper system for varying damping co-efficient. [16]

Code No: R05412101

R05**Set No. 2**

3. A uniform cantilever beam of length L has a mass per unit length m and flexural rigidity EI .
- (a) The general deflection curve of the beam can be written as
 $(x) = C_1 \sin \beta x + C_2 \cos \beta x + C_3 \sinh \beta x + C_4 \cosh \beta x$ where $\beta^4 = \omega^2 m / EI$.
 Write down the necessary boundary conditions and obtain the characteristic equation to determine the natural frequencies.
- (b) The cantilever beam can also be represented as an equivalent generalized single degree of freedom system. Write down the expressions for the generalized mass and generalized stiffness in terms of the assumed shape function. Assuming the shape function as $\psi(x) = 3x^2/2L^2 - x^3/2L^3$ obtain the values of the generalized mass and stiffness and determine the fundamental natural frequency of the system. Calculate the percentage error if the exact value is obtained as $\omega = \frac{3.516}{L^2} \sqrt{\frac{EI}{m}}$. [6+10]
4. Draw first and second modes of vibrations of the followings:
- (a) Fixed-fixed beam
 (b) Cantilever beam
 (c) Fixed-fixed string
 (d) String fixed at one end and supported by a spring at the other end
 (e) Pinned - pinned beam. [16]
5. Using a two-degree-of-freedom model of an airfoil section for pitch and plunge motions and assuming steady flow aerodynamics, derive the equations of motion in terms of the lift and moment. Assuming a simple harmonic motion for classical flutter analysis, obtain the flutter determinant condition for a non-trivial solution. [16]
6. A projectile of mass $m = 10\text{kg}$, travelling at velocity $v = 100\text{ m/s}$, hits a board attached to a dashpot of damping co-efficient $c = 400\text{ Ns/m}$, and a spring of stiffness $k = 400000\text{ N/m}$ as shown in the Figure 7. The projectile becomes embedded in the board (of negligible mass) after hitting it.

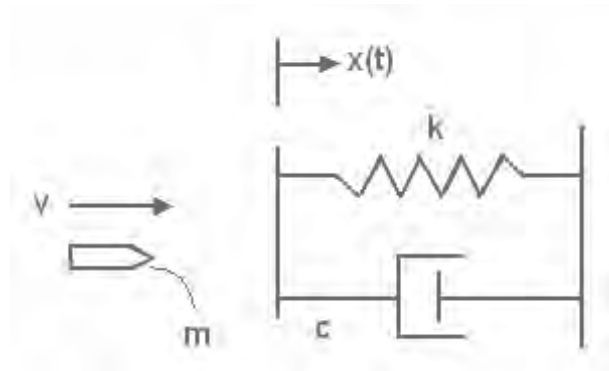


Figure 7

Answer the followings:

Code No: R05412101

R05**Set No. 2**

- (a) Derive the equation of motion of the system after the projectile hits the board
 (b) Determine the initial condition as a result of projectile hitting the board, assuming that the board-dashpot-spring system is at rest initially.
 (c) Determine the natural frequency of vibration of the motion after the projectile hits the board.
 (d) The maximum displacement of the board
 (e) The time required for the board to reach the maximum displacement. [16]
7. A uniform massless cantilever beam of total length 1.5m and $EI = 0.1 \times 10^6 \text{ Nm}^2$ carries three lumped masses at points along its length as shown in the figure 8b. The smallest mass $m_1 = 100 \text{ kg}$.
- (a) Explain the Myklestad's method for calculating the natural frequencies of this system in pure flexure with the help of the necessary equations.
 (b) Obtain the fundamental natural frequency of this system starting with an initial estimate as $w = 10 \text{ rad/s}$. [6+10]

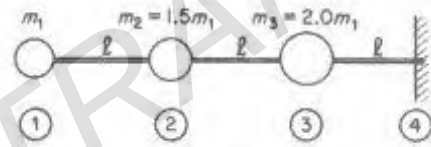


Figure 8b

8. Define whirling of a rotating shaft. Mention a few reasons for the occurrence of whirling.
 A shaft carrying a rotor of 45.36kg and eccentricity 0.254 cm rotates at 1200 rpm. Determine:
- (a) the steady-state whirl amplitude and
 (b) the maximum whirl amplitude during start-up conditions of the system. Assume the stiffness of the shaft as $3.57 \times 10^4 \text{ kg/cm}$ and the external damping as 0.1. [16]

Code No: R05412101

R05**Set No. 4**

IV B.Tech I Semester Examinations, November 2010
THEORY OF VIBRATIONS AND AEROELASTICITY
Aeronautical Engineering

Time: 3 hours**Max Marks: 80**

Answer any FIVE Questions
All Questions carry equal marks

1. Define whirling of a rotating shaft. Mention a few reasons for the occurrence of whirling.
 A shaft carrying a rotor of 45.36kg and eccentricity 0.254 cm rotates at 1200 rpm. Determine:
 - (a) the steady-state whirl amplitude and
 - (b) the maximum whirl amplitude during start-up conditions of the system. Assume the stiffness of the shaft as 3.57×10^4 kg/cm and the external damping as 0.1. [16]
2. Draw first and second modes of vibrations of the followings:
 - (a) Fixed-fixed beam
 - (b) Cantilever beam
 - (c) Fixed-fixed string
 - (d) String fixed at one end and supported by a spring at the other end
 - (e) Pinned - pinned beam. [16]
3. Describe the following:
 - (a) D'Alembert's principle
 - (b) Various forms of damping
 - (c) Principle of virtual work
 - (d) Amplitude vs Frequency response of a spring-mass-damper system for varying damping co-efficient. [16]
4. Figure below shows an equivalent model for a two storey building. The system consists of two rigid masses m_1 and m_2 supported on elastic columns. Columns as massless beams supported at both the ends. Elastic properties of the columns are as shown in the figure 6, Assume that the building only goes under horizontal motion. Let $m_1 = m_2 = m$, $H_1 = H_2 = H$, $I_1 = I_2 = I$.

Code No: R05412101

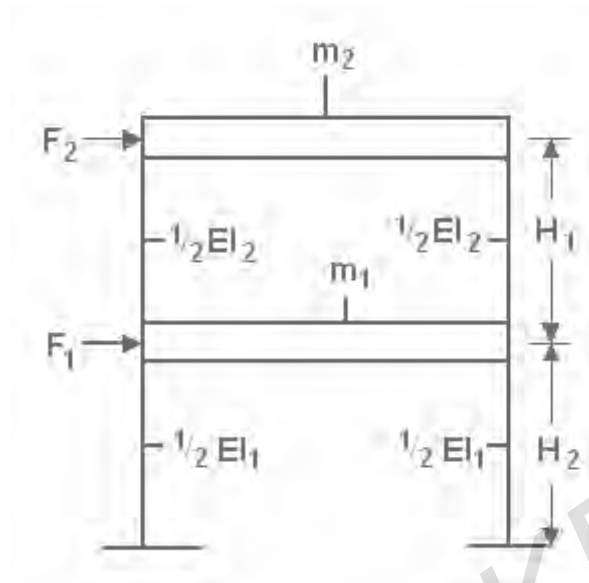
R05**Set No. 4**

Figure 6

- (a) Derive the differential equations for the horizontal motion of the masses
 - (b) Obtain the natural frequencies of the system
 - (c) Obtain the natural modes of vibration
 - (d) Plot the modes. [16]
5. A uniform massless cantilever beam of total length 1.5m and $EI = 0.1 \times 10^6 \text{ Nm}^2$ carries three lumped masses at points along its length as shown in the figure 8b. The smallest mass $m_1 = 100 \text{ kg}$.
- (a) Explain the Myklestad's method for calculating the natural frequencies of this system in pure flexure with the help of the necessary equations.
 - (b) Obtain the fundamental natural frequency of this system starting with an initial estimate as $w = 10 \text{ rad/s}$. [6+10]

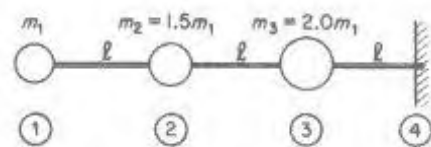


Figure 8b

6. A projectile of mass $m = 10 \text{ kg}$, travelling at velocity $v = 100 \text{ m/s}$, hits a board attached to a dashpot of damping co-efficient $c = 400 \text{ Ns/m}$, and a spring of stiffness $k = 400000 \text{ N/m}$ as shown in the Figure 7. The projectile becomes embedded in the board (of negligible mass) after hitting it.

Code No: R05412101

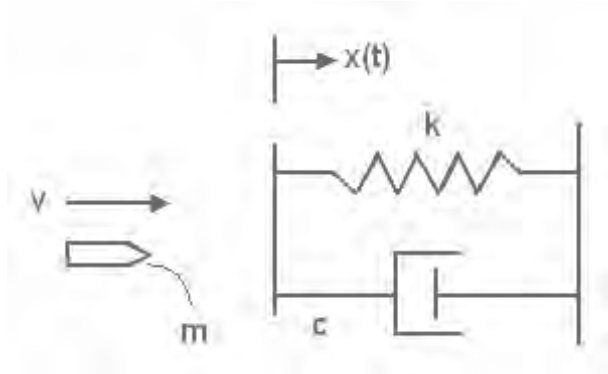
R05**Set No. 4**

Figure 7

Answer the followings:

- (a) Derive the equation of motion of the system after the projectile hits the board
 - (b) Determine the initial condition as a result of projectile hitting the board, assuming that the board-dashpot-spring system is at rest initially.
 - (c) Determine the natural frequency of vibration of the motion after the projectile hits the board.
 - (d) The maximum displacement of the board
 - (e) The time required for the board to reach the maximum displacement. [16]
7. A uniform cantilever beam of length L has a mass per unit length m and flexural rigidity EI .
- (a) The general deflection curve of the beam can be written as

$$(x) = C_1 \sin \beta x + C_2 \cos \beta x + C_3 \sinh \beta x + C_4 \cosh \beta x$$
 where $\beta^4 = \omega^2 m / EI$. Write down the necessary boundary conditions and obtain the characteristic equation to determine the natural frequencies.
 - (b) The cantilever beam can also be represented as an equivalent generalized single degree of freedom system. Write down the expressions for the generalized mass and generalized stiffness in terms of the assumed shape function. Assuming the shape function as $\psi(x) = 3x^2/2L^2 - x^3/2L^3$ obtain the values of the generalized mass and stiffness and determine the fundamental natural frequency of the system. Calculate the percentage error if the exact value is obtained as $\omega = \frac{3.516}{L^2} \sqrt{\frac{EI}{m}}$. [6+10]
8. Using a two-degree-of-freedom model of an airfoil section for pitch and plunge motions and assuming steady flow aerodynamics, derive the equations of motion in terms of the lift and moment. Assuming a simple harmonic motion for classical flutter analysis, obtain the flutter determinant condition for a non-trivial solution. [16]

Code No: R05412101

R05**Set No. 1**

IV B.Tech I Semester Examinations, November 2010
THEORY OF VIBRATIONS AND AEROELASTICITY
Aeronautical Engineering

Time: 3 hours

Max Marks: 80

Answer any FIVE Questions
 All Questions carry equal marks

1. A uniform massless cantilever beam of total length 1.5m and $EI = 0.1 \times 10^6 \text{ Nm}^2$ carries three lumped masses at points along its length as shown in the figure 8b. The smallest mass $m_1 = 100 \text{ kg}$.

- (a) Explain the Myklestad's method for calculating the natural frequencies of this system in pure flexure with the help of the necessary equations.
 (b) Obtain the fundamental natural frequency of this system starting with an initial estimate as $\omega = 10 \text{ rad/s}$. [6+10]

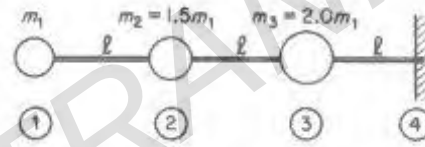


Figure 8b

2. Describe the following:
 (a) D'Alembert's principle
 (b) Various forms of damping
 (c) Principle of virtual work
 (d) Amplitude vs Frequency response of a spring-mass-damper system for varying damping co-efficient. [16]

3. A uniform cantilever beam of length L has a mass per unit length m and flexural rigidity EI .

- (a) The general deflection curve of the beam can be written as

$$(x) = C_1 \sin \beta x + C_2 \cos \beta x + C_3 \sinh \beta x + C_4 \cosh \beta x$$
 where $\beta^4 = \omega^2 m / EI$. Write down the necessary boundary conditions and obtain the characteristic equation to determine the natural frequencies.
 (b) The cantilever beam can also be represented as an equivalent generalized single degree of freedom system. Write down the expressions for the generalized mass and generalized stiffness in terms of the assumed shape function. Assuming the shape function as $\psi(x) = 3x^2/2L^2 - x^3/2L^3$ obtain the values of the generalized mass and stiffness and determine the fundamental natural frequency of the system. Calculate the percentage error if the exact value is obtained as $\omega = \frac{3.516}{L^2} \sqrt{\frac{EI}{m}}$. [6+10]

Code No: R05412101

R05**Set No. 1**

4. Figure below shows an equivalent model for a two storey building. The system consists of two rigid masses m_1 and m_2 supported on elastic columns. Columns as massless beams supported at both the ends. Elastic properties of the columns are as shown in the figure 6, Assume that the building only goes under horizontal motion. Let $m_1 = m_2 = m$, $H_1 = H_2 = H$, $I_1 = I_2 = I$.

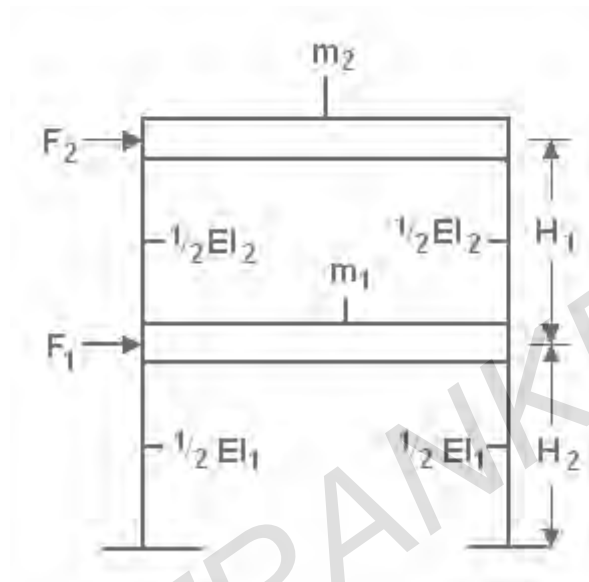


Figure 6

- Derive the differential equations for the horizontal motion of the masses
 - Obtain the natural frequencies of the system
 - Obtain the natural modes of vibration
 - Plot the modes. [16]
5. Draw first and second modes of vibrations of the followings:
- Fixed-fixed beam
 - Cantilever beam
 - Fixed-fixed string
 - String fixed at one end and supported by a spring at the other end
 - Pinned - pinned beam. [16]
6. Define whirling of a rotating shaft. Mention a few reasons for the occurrence of whirling.
A shaft carrying a rotor of 45.36kg and eccentricity 0.254 cm rotates at 1200 rpm. Determine:
- the steady-state whirl amplitude and
 - the maximum whirl amplitude during start-up conditions of the system. Assume the stiffness of the shaft as 3.57×10^4 kg/cm and the external damping as 0.1. [16]

Code No: R05412101

R05**Set No. 1**

7. Using a two-degree-of-freedom model of an airfoil section for pitch and plunge motions and assuming steady flow aerodynamics, derive the equations of motion in terms of the lift and moment. Assuming a simple harmonic motion for classical flutter analysis, obtain the flutter determinant condition for a non-trivial solution. [16]
8. A projectile of mass $m = 10\text{kg}$, travelling at velocity $v = 100\text{ m/s}$, hits a board attached to a dashpot of damping co-efficient $c = 400\text{ Ns/m}$, and a spring of stiffness $k = 400000\text{ N/m}$ as shown in the Figure 7. The projectile becomes embedded in the board (of negligible mass) after hitting it.

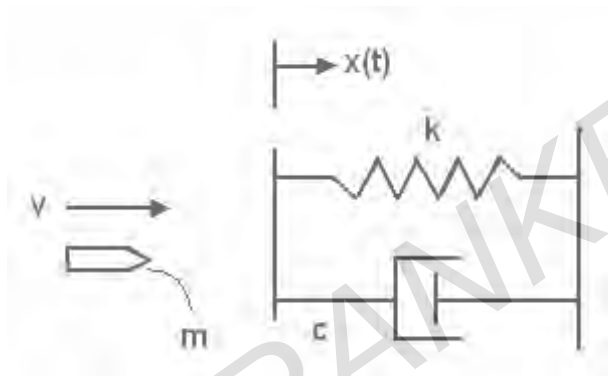


Figure 7

Answer the followings:

- Derive the equation of motion of the system after the projectile hits the board
- Determine the initial condition as a result of projectile hitting the board, assuming that the board-dashpot-spring system is at rest initially.
- Determine the natural frequency of vibration of the motion after the projectile hits the board.
- The maximum displacement of the board
- The time required for the board to reach the maximum displacement. [16]

Code No: R05412101

R05**Set No. 3**

IV B.Tech I Semester Examinations, November 2010
THEORY OF VIBRATIONS AND AEROELASTICITY
Aeronautical Engineering

Time: 3 hours

Max Marks: 80

Answer any FIVE Questions
 All Questions carry equal marks

- Using a two-degree-of-freedom model of an airfoil section for pitch and plunge motions and assuming steady flow aerodynamics, derive the equations of motion in terms of the lift and moment. Assuming a simple harmonic motion for classical flutter analysis, obtain the flutter determinant condition for a non-trivial solution. [16]
- Define whirling of a rotating shaft. Mention a few reasons for the occurrence of whirling.
 A shaft carrying a rotor of 45.36kg and eccentricity 0.254 cm rotates at 1200 rpm. Determine:
 - the steady-state whirl amplitude and
 - the maximum whirl amplitude during start-up conditions of the system. Assume the stiffness of the shaft as 3.57×10^4 kg/cm and the external damping as 0.1. [16]
- A uniform cantilever beam of length L has a mass per unit length m and flexural rigidity EI .
 - The general deflection curve of the beam can be written as $(x) = C_1 \sin \beta x + C_2 \cos \beta x + C_3 \sinh \beta x + C_4 \cosh \beta x$ where $\beta^4 = \omega^2 m / EI$. Write down the necessary boundary conditions and obtain the characteristic equation to determine the natural frequencies.
 - The cantilever beam can also be represented as an equivalent generalized single degree of freedom system. Write down the expressions for the generalized mass and generalized stiffness in terms of the assumed shape function. Assuming the shape function as $\psi(x) = 3x^2/2L^2 - x^3/2L^3$ obtain the values of the generalized mass and stiffness and determine the fundamental natural frequency of the system. Calculate the percentage error if the exact value is obtained as $\omega = \frac{3.516}{L^2} \sqrt{\frac{EI}{m}}$. [6+10]
- Draw first and second modes of vibrations of the followings:
 - Fixed-fixed beam
 - Cantilever beam
 - Fixed-fixed string
 - String fixed at one end and supported by a spring at the other end

Code No: R05412101

R05**Set No. 3**

(e) Pinned - pinned beam.

[16]

5. Describe the following:

- (a) D'Alembert's principle
- (b) Various forms of damping
- (c) Principle of virtual work
- (d) Amplitude vs Frequency response of a spring-mass-damper system for varying damping co-efficient. [16]

6. Figure below shows an equivalent model for a two storey building. The system consists of two rigid masses m_1 and m_2 supported on elastic columns. Columns as massless beams supported at both the ends. Elastic properties of the columns are as shown in the figure 6, Assume that the building only goes under horizontal motion. Let $m_1 = m_2 = m$, $H_1 = H_2 = H$, $I_1 = I_2 = I$.

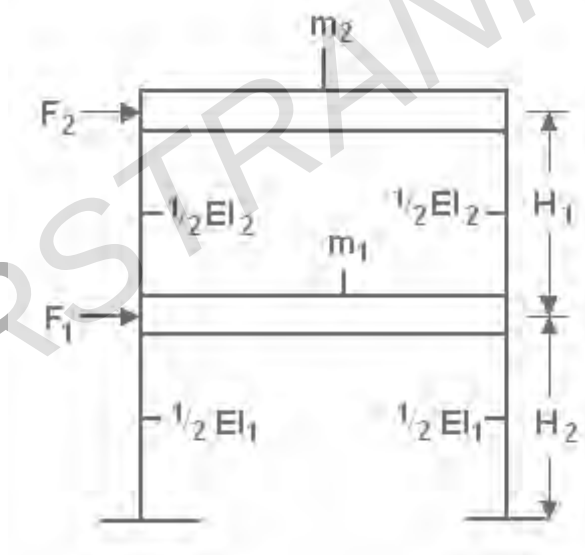


Figure 6

- (a) Derive the differential equations for the horizontal motion of the masses
- (b) Obtain the natural frequencies of the system
- (c) Obtain the natural modes of vibration
- (d) Plot the modes. [16]

7. A projectile of mass $m = 10\text{kg}$, travelling at velocity $v = 100\text{ m/s}$, hits a board attached to a dashpot of damping co-efficient $c = 400\text{ Ns/m}$, and a spring of stiffness $k = 400000\text{ N/m}$ as shown in the Figure 7. The projectile becomes embedded in the board (of negligible mass) after hitting it.

Code No: R05412101

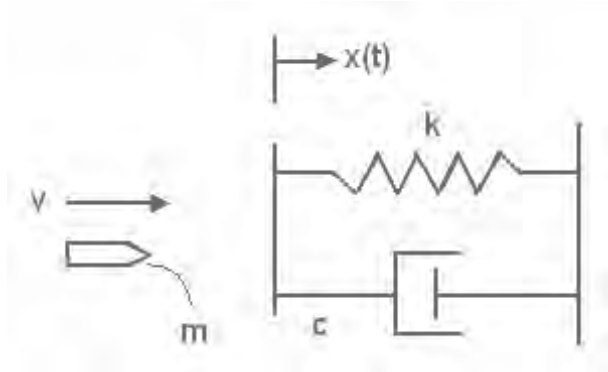
R05**Set No. 3**

Figure 7

Answer the followings:

- (a) Derive the equation of motion of the system after the projectile hits the board
 - (b) Determine the initial condition as a result of projectile hitting the board, assuming that the board-dashpot-spring system is at rest initially.
 - (c) Determine the natural frequency of vibration of the motion after the projectile hits the board.
 - (d) The maximum displacement of the board
 - (e) The time required for the board to reach the maximum displacement. [16]
8. A uniform massless cantilever beam of total length 1.5m and $EI = 0.1 \times 10^6 \text{ Nm}^2$ carries three lumped masses at points along its length as shown in the figure 8b. The smallest mass $m_1 = 100 \text{ kg}$.
- (a) Explain the Myklestad's method for calculating the natural frequencies of this system in pure flexure with the help of the necessary equations.
 - (b) Obtain the fundamental natural frequency of this system starting with an initial estimate as $\omega = 10 \text{ rad/s}$. [6+10]

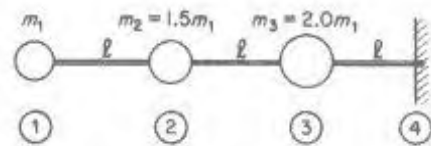


Figure 8b
